

Institute of Air-handling and Refrigeration (ILK Dresden)

**Development of a small capacity
directly air-cooled water/LiBr absorption chiller**

Chinese Solar Cooling Conference, Shanghai, 27.3.2015



- ▶ **Founded in 1964**
- ▶ **Re-established as independent research institute in 1991**

- ▶ **Employees: 145**
- ▶ **Academics: 72 %**
- ▶ **mean age: ~44**
- ▶ **Laboratory area: ~3000 m²**
- ▶ **Test rigs: ~56**
- ▶ **Phys. / Chem. Laboratories: 25**





Problems in solar thermal cooling systems with small scale chillers

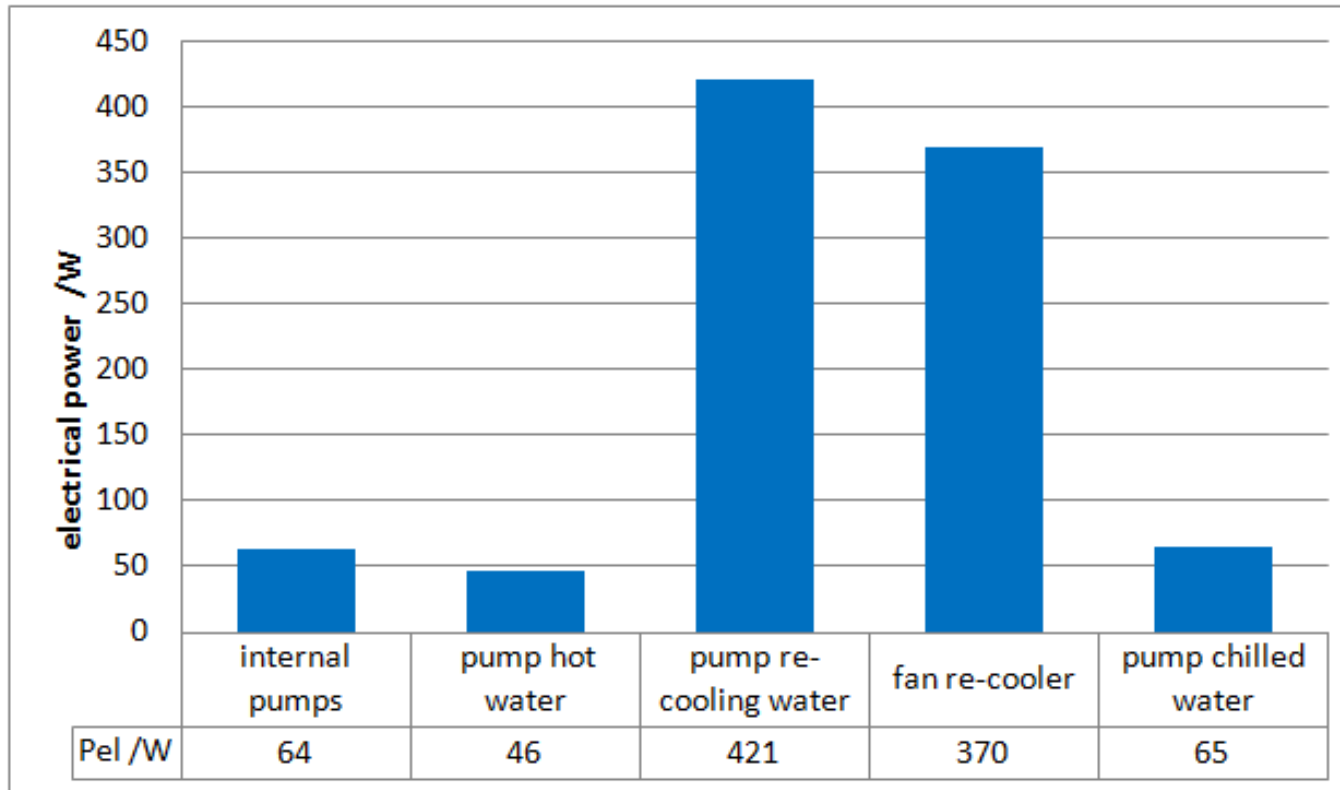
- ▶ **Complexity of the system**
- ▶ **Interface problems because of different crafts (might be) involved**
- ▶ **Possibly high error rate during installation**
- ▶ **Auxiliary energy demand of the system**
- ▶ **Limited applicability of evaporative systems but high re-cooling sensibility of the cycle**



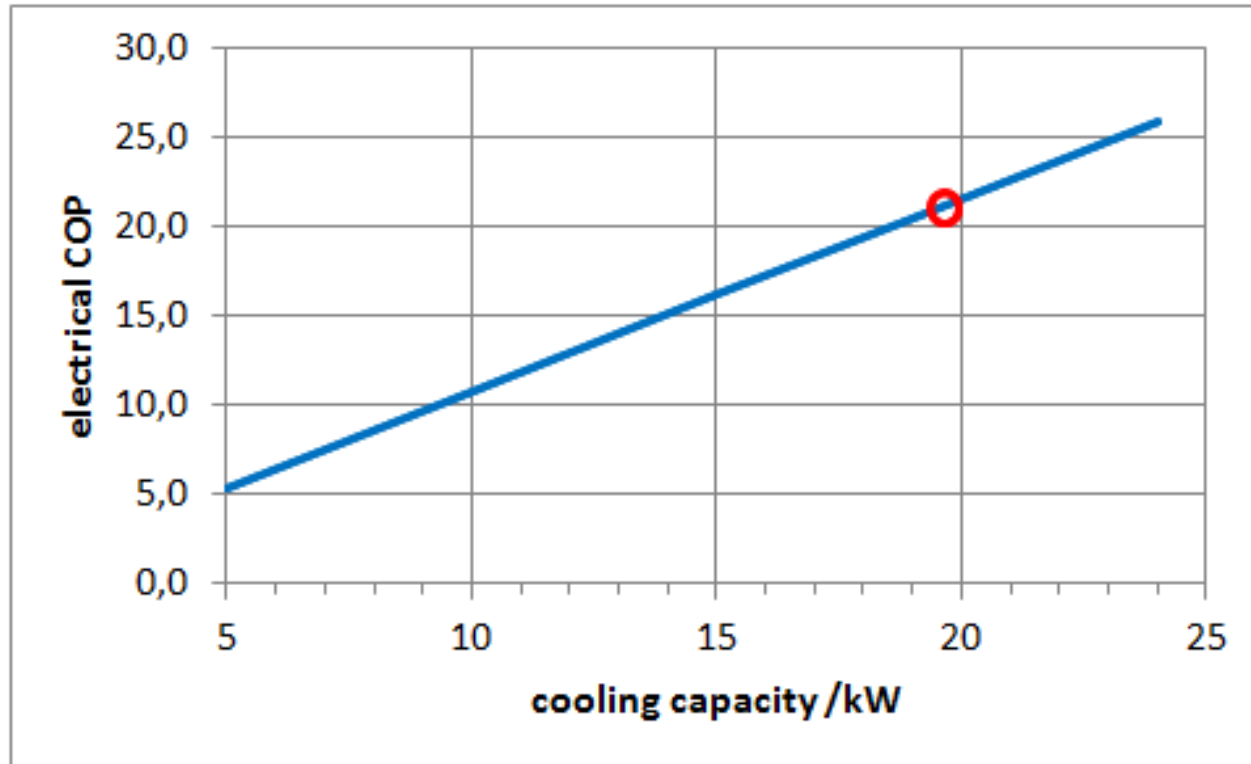
- ▶ **Avoiding of an intermediate circuit:**
 - to minimize the auxiliary energy demand (as re-cooling circuit pump usually consumes much electricity)
 - for a better approach of external and internal temperatures since temperature lift and driving temperature are limited
- ▶ **Air-cooled absorber needed**
- ▶ **Water as refrigerant -> big free section needed**
- ▶ **Air as heat transfer media -> big free section needed**



Distribution of the auxiliary energy demand of a small scale water cooled absorption chiller

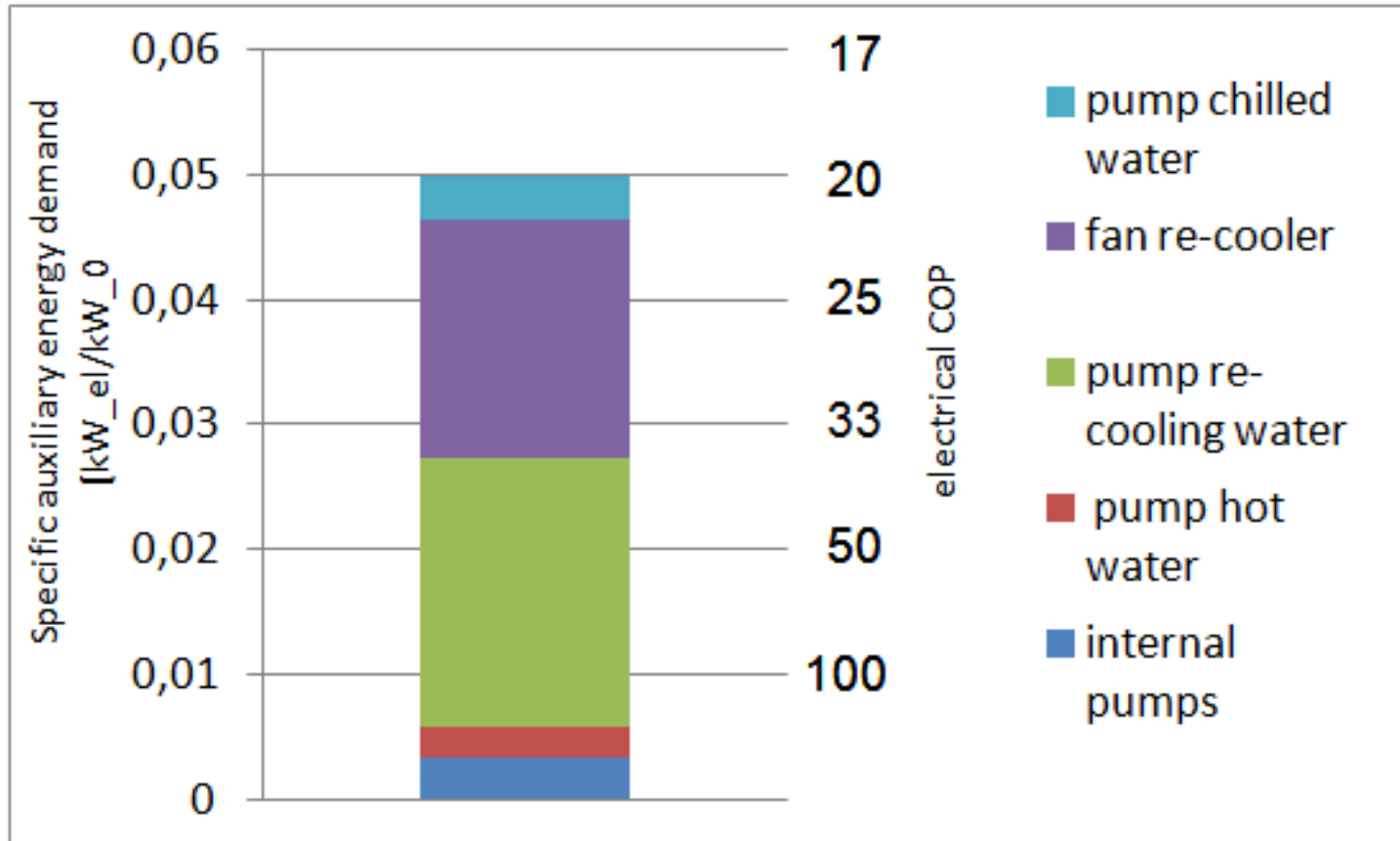


Nominal cooling capacity: 19,4 kW



Auxiliary energy demand of the system with constant speed pumps/fans as a function the cooling capacity

Distribution of auxiliary energy demand



Examples of air-cooled absorption chillers



Source: Rotartica

$\text{H}_2\text{O}/\text{LiBr}$, rotating HX,
intermediate circuit



Source Helioplus

$\text{NH}_3/\text{H}_2\text{O}$ -> high working pressure!
Gas driven -> high driving temperature

Examples of air-cooled absorption chillers



Source: Broad

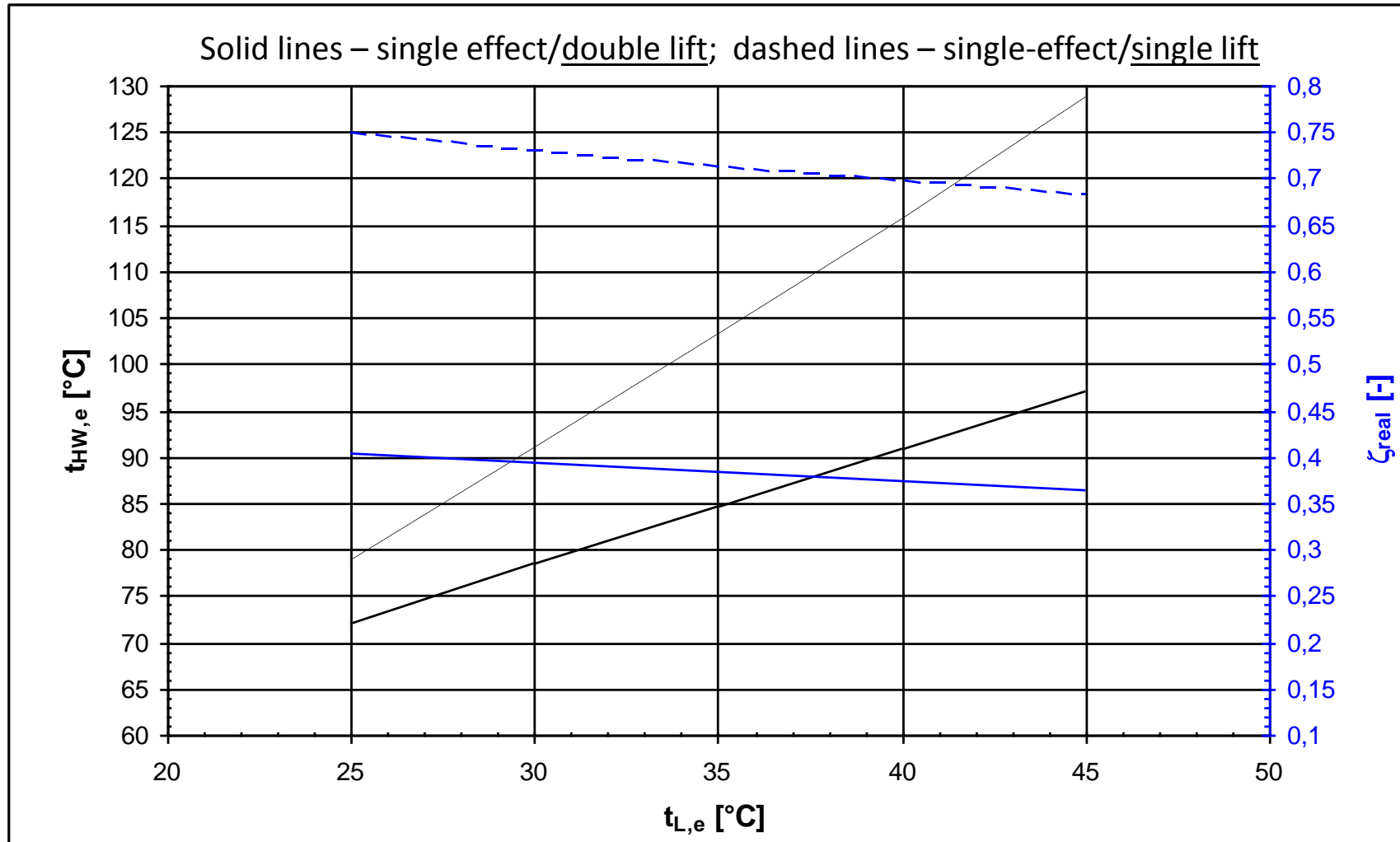


Source: Broad

H₂O/LiBr, gas driven, intermediate circuit

**Q₀=23 kW; zetta=1,1, P_{el}=1,8kW
(COP_{el}=12,7)**

Balancing Single Lift vs. Double Lift



chilled water out: 13 °C



Aimed specifications for the development

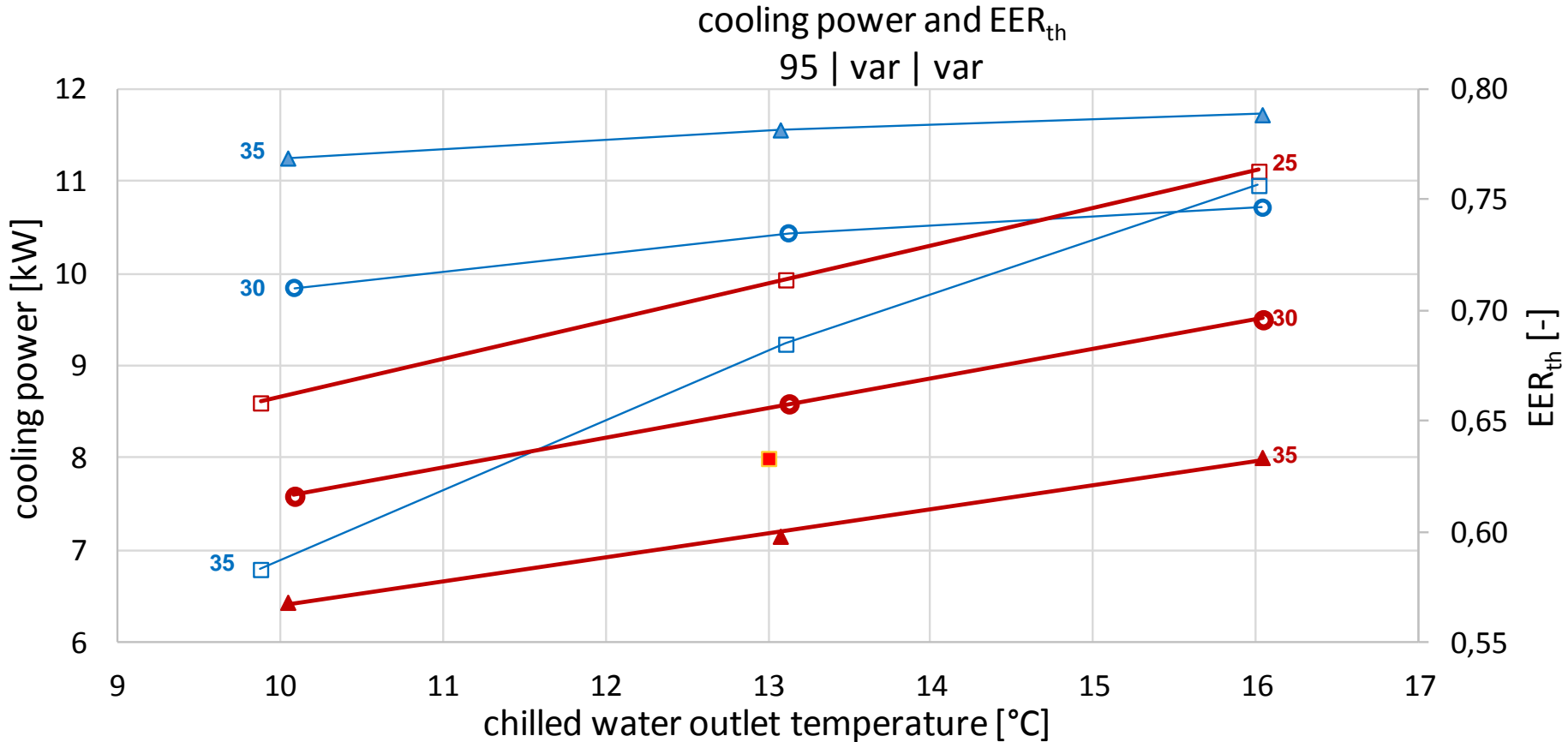
<u>External Fluid</u>	<u>Nominal Condition</u>	<u>Operating Range</u>
Chilled water temperature (water w. 20 % Glycol)	18 °C / 13 °C (in/out)	6 °C ... 20 °C (out)
Heating water temperature (water w. 20 % Glycol)	95 °C / 87 °C (in/out)	75 °C ... 105 °C (in)
Ambient air (for re-cooling)	32 °C / 42 °C (in/out)	10 °C ... 32°C (in)
Cooling capacity	8 kW	

- ▶ Condenser and Absorber directly air-cooled
- ▶ Outdoor installation, frost save
- ▶ Auxiliary energy consumption at nominal conditions $< 60 W_{el}/kW_0$ („EER“ > 16)
- ▶ Single effect / single lift

Test of components within functional model







- 95 | 25 | var
- 95 | 30 | var
- ▲ 95 | 35 | var
- design point (95 | 32 | 13)

- EER_{th} 95 | 25 | var
- EER_{th} 95 | 30 | var
- ▲ EER_{th} 95 | 35 | var

Restrictions of lab measurements

- ▶ Limited space and height in lab
- ▶ No free outflow of absorption chiller outlet air
 - > increased air-side counter pressure
 - > higher fan speed for same air flow rate needed
 - > increased power consumption of fan
 - => no meaningful results regarding EER_{el}
- ▶ Difficulty to maintain even temperature distribution





- ▶ **Thermal design point reached**
- ▶ **No useful results regarding electrical efficiency yet**
- ▶ **Field test planned in summer 2015**

Results expected regarding

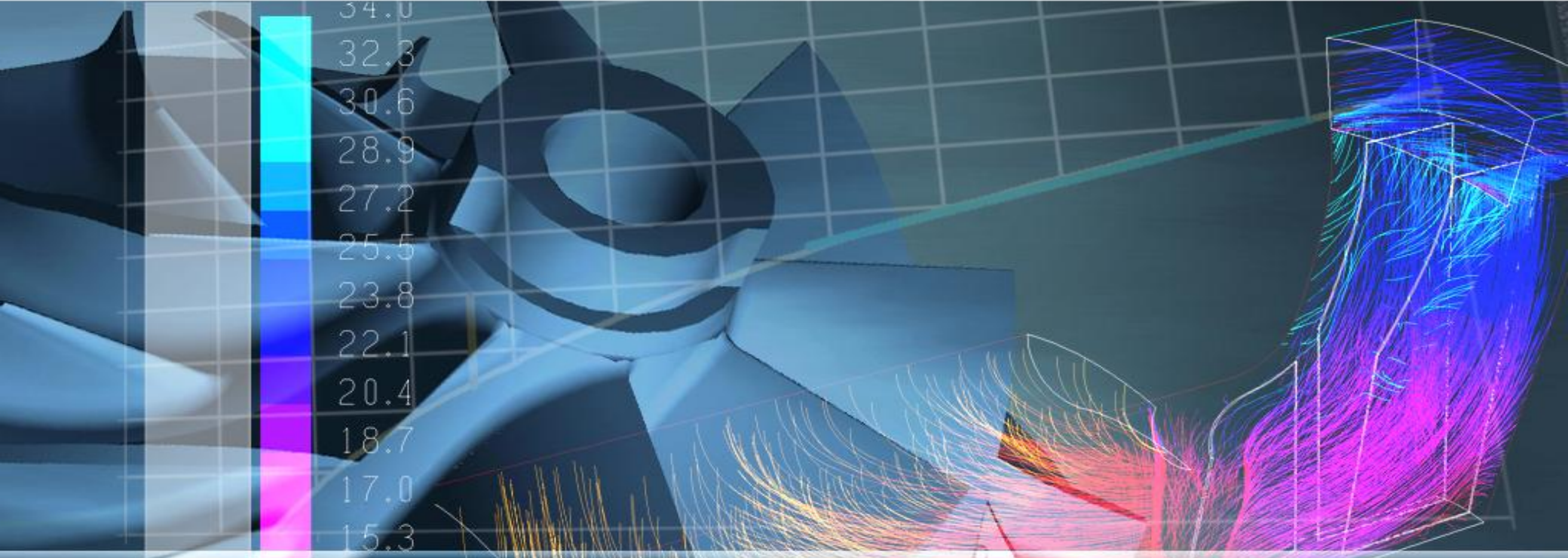
- ▶ **Electrical efficiency**
- ▶ **Operational experience**
- ▶ **Outdoor installation**

Vacuum ice slurry technology for high density cold storage, e.g. in PV driven cooling systems



- ▶ High energy density (93 kWh/m³)
- ▶ Higher efficiency than conventional ice bank storage through high evaporation temperature
- ▶ No glycol circuit needed
- ▶ Ice slurry is pumpable
- ▶ Capacity range: 50 ... 500 kW
- ▶ Matching solar radiation and cooling demand





Thanks for your attention!

Questions?

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